CHAPTER 1 FUNDAMENTAL CONCEPTS

1.1 We use the first three steps of Eq. 1.11

$$\varepsilon_{x} = \frac{\sigma_{x}}{E} - v \frac{\sigma_{y}}{E} - v \frac{\sigma_{z}}{E}$$
$$\varepsilon_{y} = -v \frac{\sigma_{x}}{E} + \frac{\sigma_{y}}{E} - v \frac{\sigma_{z}}{E}$$
$$\varepsilon_{z} = -v \frac{\sigma_{x}}{E} - v \frac{\sigma_{y}}{E} + \frac{\sigma_{z}}{E}$$

Adding the above, we get

$$\varepsilon_x + \varepsilon_y + \varepsilon_z = \frac{1 - 2\nu}{E} (\sigma_x + \sigma_y + \sigma_z)$$

Adding and subtracting $v \frac{\sigma_x}{E}$ from the first equation,

$$\varepsilon_{x} = \frac{1+\nu}{E}\sigma_{x} - \frac{\nu}{E}(\sigma_{x} + \sigma_{y} + \sigma_{z})$$

Similar expressions can be obtained for ε_y , and ε_z .

From the relationship for γ_{yz} and Eq. 1.12,

$$f_{yz} = \frac{E}{2(1+\nu)}\gamma_{yz}$$
 etc.

Above relations can be written in the form

 $\sigma = D\epsilon$

where \mathbf{D} is the material property matrix defined in Eq. 1.15.

1.2 Note that $u_2(x)$ satisfies the zero slope boundary condition at the support.



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1.3 Plane strain condition implies that $\varepsilon_z = 0 = -v \frac{\sigma_x}{F} - v \frac{\sigma_y}{F} + \frac{\sigma_z}{F}$ which gives $\sigma_z = v(\sigma_x + \sigma_y)$ We have, $\sigma_x = 20000 \text{ psi}$ $\sigma_y = -10000 \text{ psi}$ $E = 30 \times 10^6 \text{ psi}$ v = 0.3. On substituting the values, $\sigma_{z} = 3000 \text{ psi}$ 1.4 Displacement field $u = 10^{-4} (-x^2 + 2y^2 + 6xy)$ $v = 10^{-4} (3x + 6y - y^2)$ $\frac{\partial u}{\partial x} = 10^{-4} \left(-2x + 6y \right) \quad \frac{\partial u}{\partial y} = 10^{-4} \left(4y + 6x \right)$ $\frac{\partial v}{\partial x} = 3 \times 10^{-4} \qquad \qquad \frac{\partial v}{\partial y} = 10^{-4} \left(6 + 2y \right)$ $\varepsilon = \begin{cases} \frac{\partial u}{\partial x} \\ \frac{\partial v}{\partial y} \\ \frac{\partial u}{\partial y} + \frac{\partial v}{\partial x} \end{cases}$ at x = 1, y = 0 $\varepsilon = 10^{-4} \begin{cases} -2\\ 6\\ 0 \end{cases}$ **1.5** On inspection, we note that the displacements u and v are given by u = 0.1 y + 4v = 0It is then easy to see that htroduction to Finite Elements in Engineering, Fourth Edition, by T. R. Chandrupatla and A. D. Belegundu. ISBN 01-3-216274-1.

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$$\varepsilon_x = \frac{\partial u}{\partial x} = 0$$

$$\varepsilon_y = \frac{\partial v}{\partial y} = 0$$

$$\gamma_{xy} = \frac{\partial u}{\partial y} + \frac{\partial v}{\partial x} = 0.1$$

1.6 The displacement field is given as

$$u = 1 + 3x + 4x^{3} + 6xy^{2}$$

$$v = xy - 7x^{2}$$

(a) The strains are then given by

$$\varepsilon_x = \frac{\partial u}{\partial x} = 3 + 12x^2 + 6y^2$$
$$\varepsilon_y = \frac{\partial v}{\partial y} = x$$
$$\gamma_{xy} = \frac{\partial u}{\partial y} + \frac{\partial v}{\partial x} = 12xy + y - 14x$$

(b) In order to draw the contours of the strain field using MATLAB, we need to create a script file, which may be edited as a text file and save with ".m" extension. The file for plotting ε_x is given below

```
file "prob1p5b.m"
    [X,Y] = meshgrid(-1:.1:1,-1:.1:1);
    Z = 3.+12.*X.^2+6.*Y.^2;
    [C,h] = contour(X,Y,Z);
    clabel(C,h);
```

On running the program, the contour map is shown as follows:

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1.8

$$\sigma_{x} = 40 \text{ MPa} \quad \sigma_{y} = 20 \text{ MPa} \quad \sigma_{z} = 30 \text{ MPa}$$

$$\tau_{yz} = -30 \text{ MPa} \quad \tau_{xz} = 15 \text{ MPa} \quad \tau_{xy} = 10 \text{ MPa}$$

$$\mathbf{n} = \begin{bmatrix} \frac{1}{2} & \frac{1}{2} & \frac{1}{\sqrt{2}} \end{bmatrix}^{\mathrm{T}}$$
From Eq.1.8 we get
$$T_{x} = \sigma_{x}n_{x} + \tau_{xy}n_{y} + \tau_{xz}n_{z}$$

$$\cdot = 35.607 \text{ MPa}$$

$$T_{y} = \tau_{xy}n_{x} + \sigma_{y}n_{y} + \tau_{yz}n_{z}$$

$$= -6.213 \text{ MPa}$$

$$T_{z} = \tau_{xz}n_{x} + \tau_{yz}n_{y} + \sigma_{z}n_{z}$$

$$= 13.713 \text{ MPa}$$

$$\sigma_{n} = T_{x}n_{x} + T_{y}n_{y} + T_{z}n_{z}$$

$$= 24.393 \text{ MPa}$$

1.9 From the derivation made in P1.1, we have

$$\sigma_{x} = \frac{E}{(1+\nu)(1-2\nu)} [(1-\nu)\varepsilon_{x} + \nu\varepsilon_{y} + \nu\varepsilon_{z}]$$

which can be written in the form

$$\sigma_{x} = \frac{E}{(1+\nu)(1-2\nu)} [(1-2\nu)\varepsilon_{x} + \nu\varepsilon_{\nu}]$$

and

$$\tau_{yz} = \frac{E}{2(1+\nu)}\gamma_{yz}$$

Lame's constants λ and μ are defined in the expressions

$$\sigma_x = \lambda \varepsilon_v + 2\mu \varepsilon_x$$
$$\tau = \mu \gamma$$

$$\tau_{yz} = \mu \gamma_{yz}$$

On inspection,

$$\lambda = \frac{Ev}{(1+v)(1-2v)}$$
$$\mu = \frac{E}{2(1+v)}$$

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1.10

$$\varepsilon = 1.2 \times 10^{-5}$$

$$\Delta T = 30^{\circ} C$$

$$E = 200 \text{ GPa}$$

$$\alpha = 12 \times 10^{-6} / {}^{\circ} C$$

$$\varepsilon_{0} = \alpha \Delta T = 3.6 \times 10^{-4}$$

$$\sigma = E(\varepsilon - \varepsilon_{0}) = -69.6 \text{ MPa}$$

1.11

$$\varepsilon_x = \frac{du}{dx} = 1 + 2x^2$$
$$\delta = \int_0^L \frac{du}{dx} dx = \left(x + \frac{2}{3}x^3\right)\Big|_0^L$$
$$= L\left(1 + \frac{2}{3}L^2\right)$$

1.12 Following the steps of Example 1.1, we have

$$\begin{bmatrix} (80+40+50) & -80 \\ -80 & 80 \end{bmatrix} \begin{bmatrix} q_1 \\ q_2 \end{bmatrix} = \begin{bmatrix} 60 \\ 50 \end{bmatrix}$$

Above matrix form is same as the set of equations:

 $170 q_1 - 80 q_2 = 60$ $- 80 q_1 + 80 q_2 = 50$

Solving for q_1 and q_2 , we get

$$q_1 = 1.222 \text{ mm}$$

$$q_2 = 1.847 \text{ mm}$$

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When the wall is smooth, $\sigma_x = 0$. ΔT is the temperature rise.

a) When the block is thin in the *z* direction, it corresponds to plane stress condition. The rigid walls in the *y* direction require $\varepsilon_y = 0$. The generalized Hooke's law yields the equations

$$\varepsilon_{x} = -v \frac{\sigma_{y}}{E} + \alpha \Delta T$$
$$\varepsilon_{y} = \frac{\sigma_{y}}{E} + \alpha \Delta T$$

From the second equation, setting $\varepsilon_y = 0$, we get $\sigma_y = -E\alpha\Delta T$. ε_x is then calculated using the first equation as $(1-\nu)\alpha\Delta T$.

b) When the block is very thick in the *z* direction, plain strain condition prevails. Now we have $\varepsilon_z = 0$, in addition to $\varepsilon_y = 0$. σ_z is not zero.

$$\varepsilon_{x} = -v \frac{\sigma_{y}}{E} - v \frac{\sigma_{z}}{E} + \alpha \Delta T$$
$$\varepsilon_{y} = \frac{\sigma_{y}}{E} - v \frac{\sigma_{z}}{E} + \alpha \Delta T = 0$$

$$\varepsilon_z = -v \frac{\sigma_y}{E} + \frac{\sigma_z}{E} + \alpha \Delta T = 0$$

From the last two equations, we get

$$\sigma_{y} = \frac{-E\alpha\Delta T}{1+\nu} \quad \sigma_{z} = -\frac{1+2\nu}{1+\nu}E\alpha\Delta T$$

 ε_x is now obtained from the first equation.

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